Consider a functional F defined by

$$F(w) = \frac{1}{2} \int_{\Omega} D\left\{ \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} \right)^2 - 2(1 - v) \left(\frac{\partial^2 w}{\partial x^2} \frac{\partial^2 w}{\partial y^2} - \left(\frac{\partial^2 w}{\partial x \partial y} \right)^2 \right) \right\} dy dx$$
$$-Pw(x_0, y_0) - \frac{1}{2} \int_{\Omega} N_x \left(\frac{\partial w}{\partial x} \right)^2 dy dx$$

where $D = Eh^3/(12(1-v^2))$ is the bending rigidity, *E* is Young's modulus, *h* is the thickness of the plate, v is Poisson's ratio, *P* is an applied point force, N_X is applied compressive traction, and (x_0, y_0) is a specified point in a quadrilateral domain Ω that is the one shown in the following figure.

Solve the minimization problem

$$\min_{v\in V_0}F(v)$$

where $V_0 = \{v \in H^2(\Omega) | v = 0 \text{ on } \partial\Omega\}$, $H^2(\Omega)$ is the linear space of all functions whose upto second derivatives are square integrable on W in the generalized sense, using the Ritz-Galerkin method.

Verify your results by any finite element methods available in CAEN, for example, by using MSC/NASTRAN or ABAQUS, by setting up appropriate constants and geometrical dimensions.

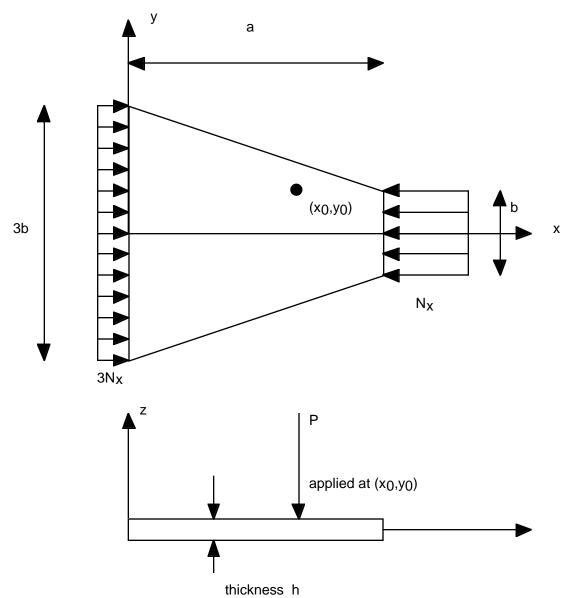


Figure 1 A Quadrilateral Plate Subjected to a Point Force P and Compressive In-plane Traction