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# **CONSTITUTIVE MODELING AND OPTIMAL DESIGN OF POLYMERIC FOAMS FOR CRASHWORTHINESS**

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# ACKNOWLEDGMENTS

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- Computational Mechanics Lab
- American Automobile Manufacturer Association
- Chrysler Motor Company
- Livermore Software Technology Corporation
- The University of Michigan ACE-MRL Lab

# Outline

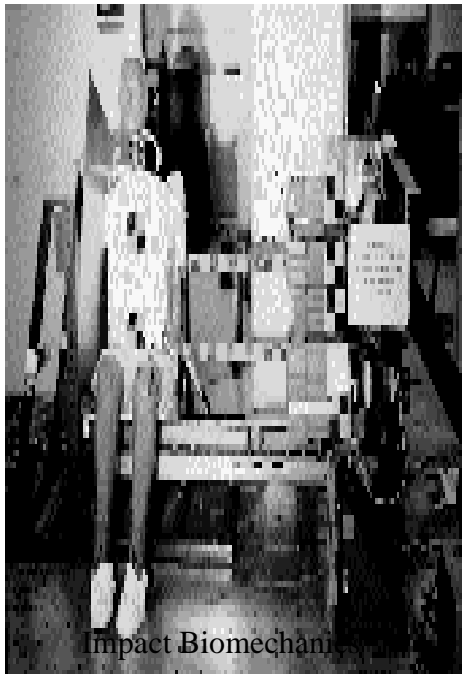
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- **Introduction**
- **Experimental Investigation and Result**
- **Constitutive Modeling**
- **Numerical Implementation Procedures**
- **Image-based Fixed-grid Homogenization Method**
- **Foam Design Optimization**
- **Conclusion and Future Work**

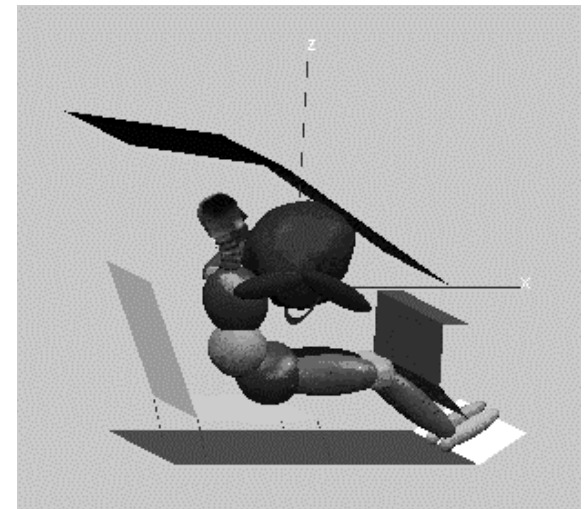
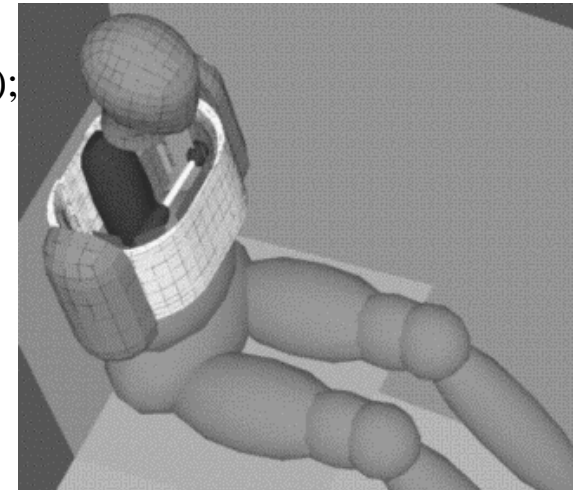
# Introduction

## ● Background

- \* In 1991, 56,000 people died in auto accident in the US (NHTSA);
- \* New federal motor vehicle safety standards (FMVSS);
- \* Usage of polymeric foam for cushion purpose;
- \* Mathematical Modeling of Transportation Safety.



Hybrid III Dummy and Honeycomb Padding



Computational Model

# Objective and Tasks

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- **Tasks**

- \* Phenomenological modeling of PU, PS and PP foams;
- \* Numerical implementation as user defined material subroutine in LS-DYNA3D;
- \* Model validation: simple loading and structural test;
- \* Microscopic constitutive modeling by image-based fixed-grid; representative volume element analysis using homogenization method
- \* Optimization of polymeric foam structure.

- **Foam specific cushion character**

- \* Limited compressive stress by long plateau regime
- \* Compression and shear properties
- \* Large deformation (80% volumetric strain) and low bulk modulus
- \* Rate sensitive: High strain rate (35 mph)
- \* Temperature sensitive: -20° C to 80° C

# Polymer Material Properties

## Types of polymer foams

(at room temperature 20° C) :

Flexible(elastomeric) foam: Polyurethane foam

Rigid (elastic-plastic) foam: Polystyrene foam

Semi rigid foam: Polypropylene foam

## Time-Temperature Correspondence

$$E_s(t, T_0) = E_s\left(\frac{t}{a_t}, T_1\right)$$

$$\log a_T = \frac{-C_1(T_1 - T_g)}{C_2 + T_1 - T_g}$$

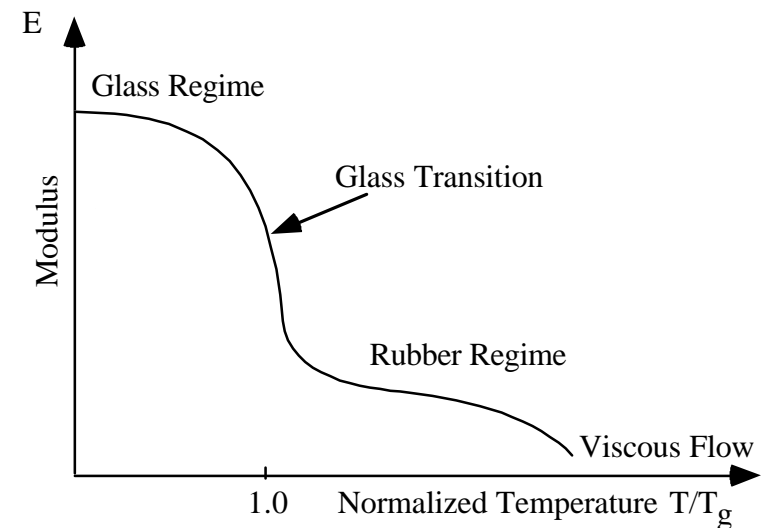


Table 1.1 Properties of Solid Polymers (at 20 °C)

Material	Density (Mg/ m <sup>3</sup> )	Glass Temperatur e (K)	Young's Modulus E <sub>s</sub> (GN/m <sup>2</sup> )	Yield Strength S <sub>ys</sub> (MN/m <sup>2</sup> )	Fracture Strength (MN/m <sup>2</sup> )	Fracture Toughness K <sub>IC</sub> (MN/m <sup>1.5</sup> )
Polyurethane	1.2	-	1.6	127	130	-
Polystyrene	1.05	373	1.2-1.7	30-70	40-80	-
Polypropylene	0.91	253	1.2-1.7	30-70	35-90	2

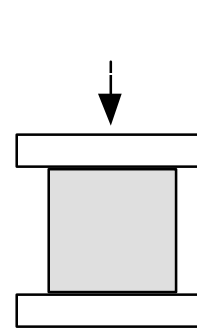
## Related Work

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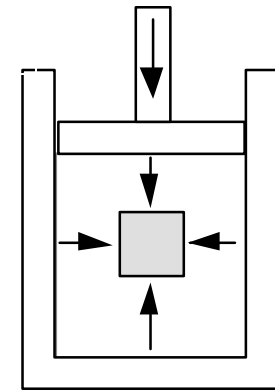
- Dimensional mechanism model
  - Gibson and Ashby (1988);
  - Gibson et al (1989) and Triantafillou et al (1989);
  - Puso and Govindjee (1995).
- Simple loading phenomenological model
  - Rush, 1969;
  - Ramon et al, 1990;
  - Sherwood and Frost, 1992.
- Continuum model
  - Roscoe's critical state theory (Schofield and Worth, 1968);
  - Krieg (1972);
  - Neilsen et al (1995).

# Experiment Program

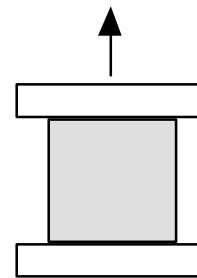
Test mode	Foam type	PP foam		PS foam		PU foam	
	Density (pcf)	1.89	3.06	1.0	4.0	4.3	6.0
	Strain rate (sec <sup>-1</sup> )						
Uniaxial Compression	1.60 x 10 <sup>-3</sup>	•	•	•	•	•	†
	8.00 x 10 <sup>-1</sup>	•	•	•	•	•	•
	4.60	•	•	•	•	•	•
	8.80 x 10 <sup>1</sup>	•	•	•	•	•	•
Hydrostatic Compression	4.00 x 10 <sup>-3</sup>	•	•			•	•
	2.00 x 10 <sup>-1</sup>	•	•			•	•
	1.15 x 10 <sup>1</sup>	•	•			•	•
Uniaxial Tension	1.60 x 10 <sup>-3</sup>					•	•
	8.00 x 10 <sup>-1</sup>					•	•
	4.60					•	•
	8.80 x 10 <sup>1</sup>						
Simple Shear	1.60 x 10 <sup>-3</sup>			•	•	•	•
	8.00 x 10 <sup>-1</sup>			•	•	•	•
	4.60			•	•	•	•
	8.80 x 10 <sup>1</sup>						



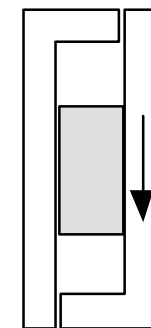
(a)



(b)



(c)



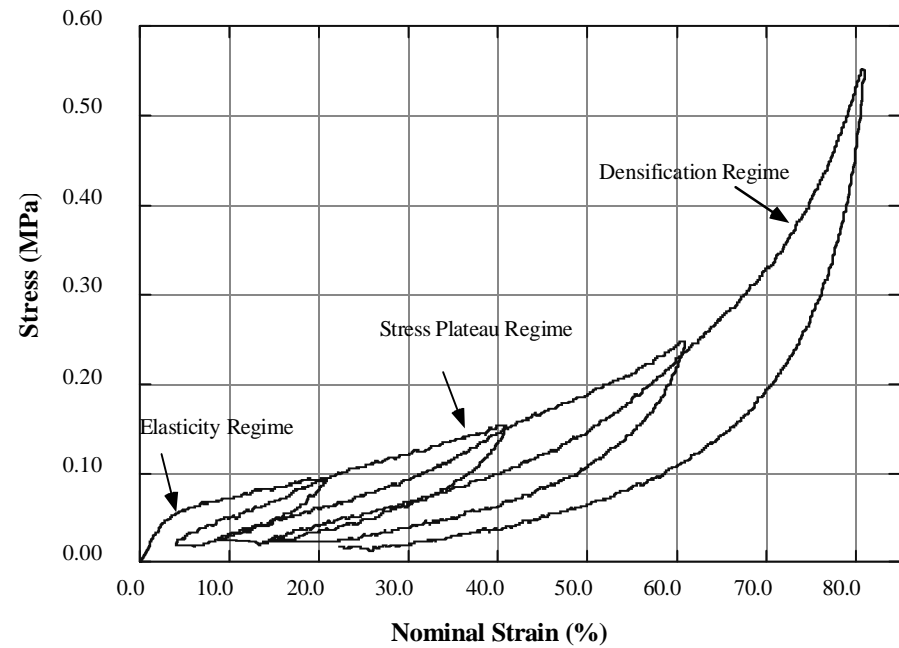
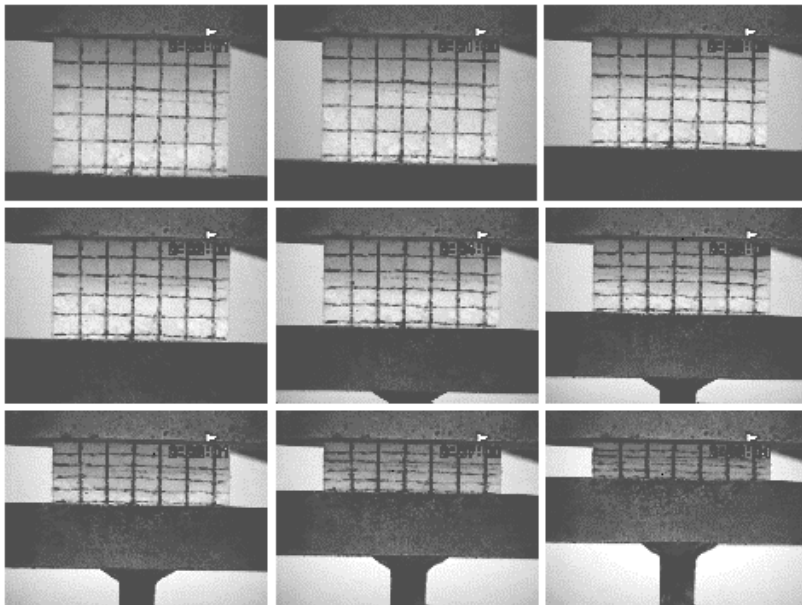
(d)

\* ASTM Standard D1621

\* 50 x 50 x 50 mm<sup>3</sup> for uniaxial and hydrostatic tests

\* 100 x 50 x 50 mm<sup>3</sup> for shear tests

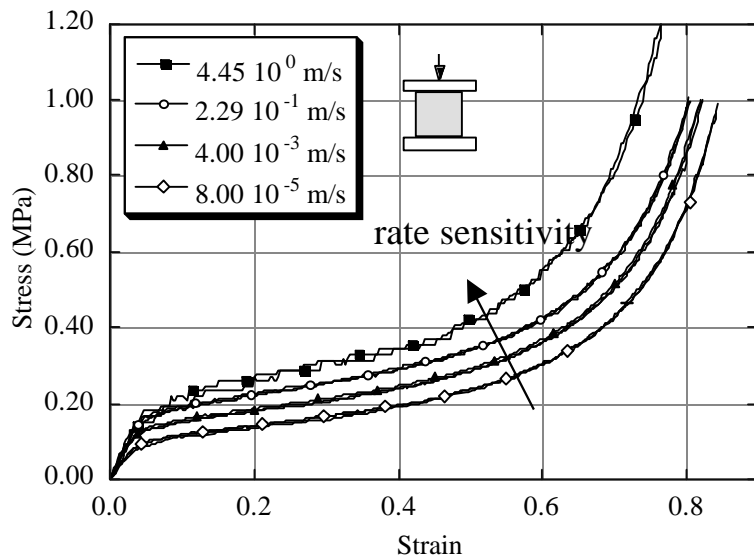
# Compressive Response of Polymeric Foam



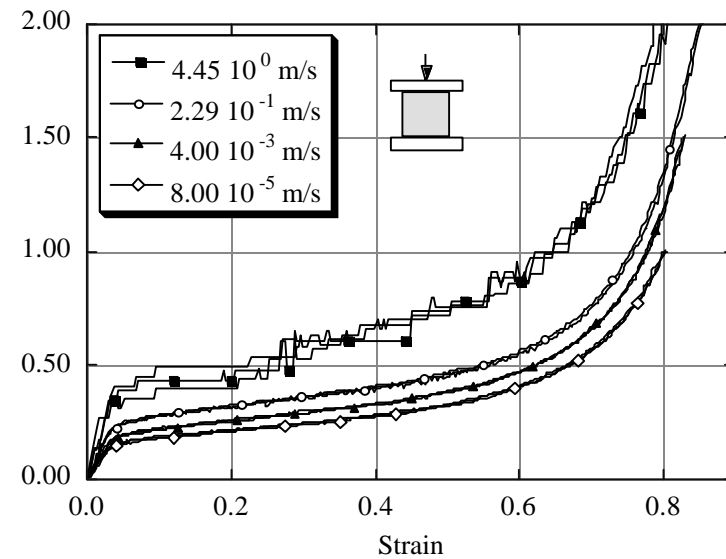
- \* Negligible size effect
- \* Uniform deformation
- \* Near zero Poisson's ratio

Quasi-static Response (BASF Polypropylene foam, 1.89 pcf)

# Compressive Responses of Polypropylene Foams

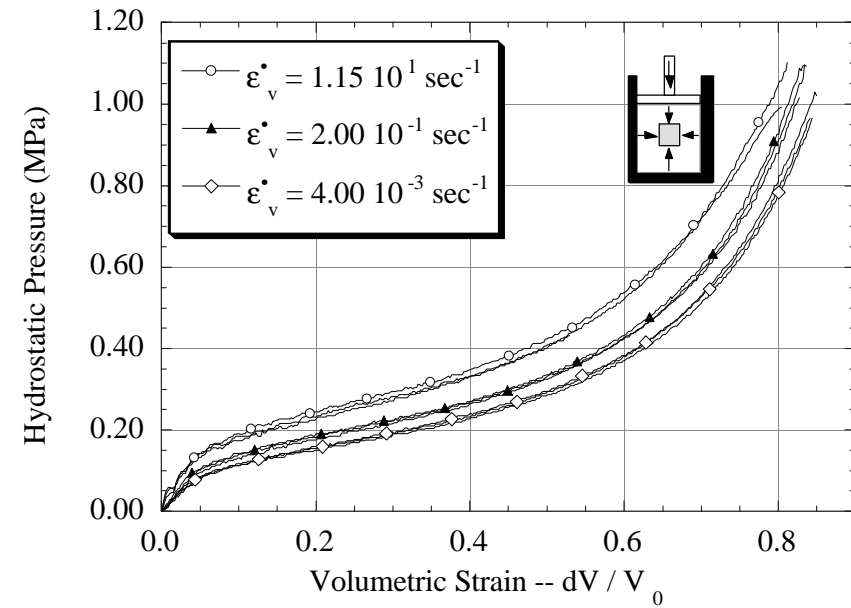
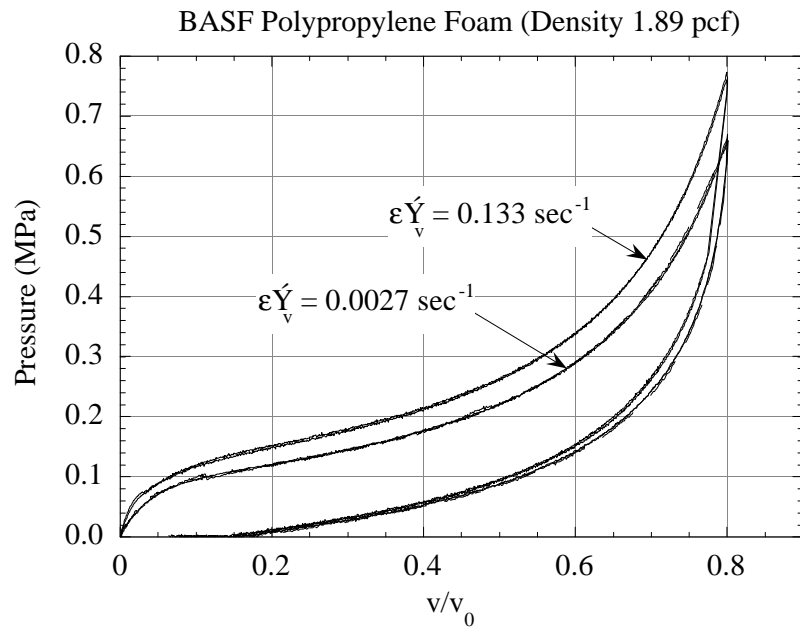


Polypropylene foam (1.89 pcf)



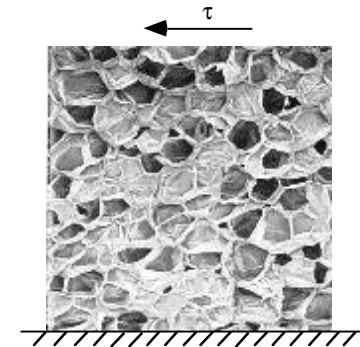
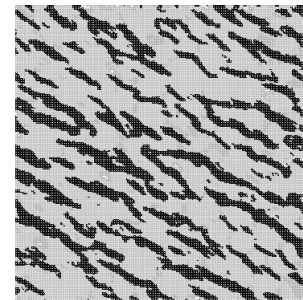
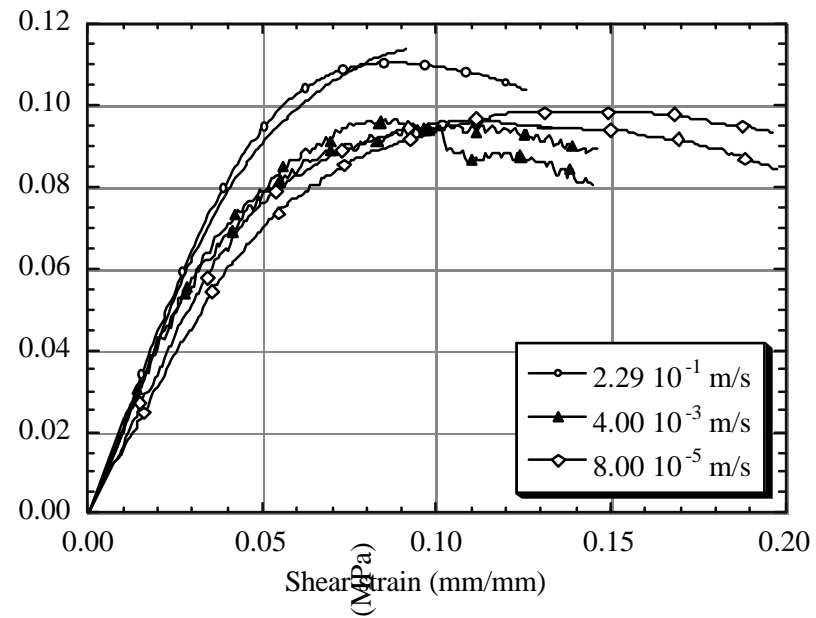
Polypropylene foam (3.06 pcf)

# Hydrostatic Compression Response of Polypropylene Foam



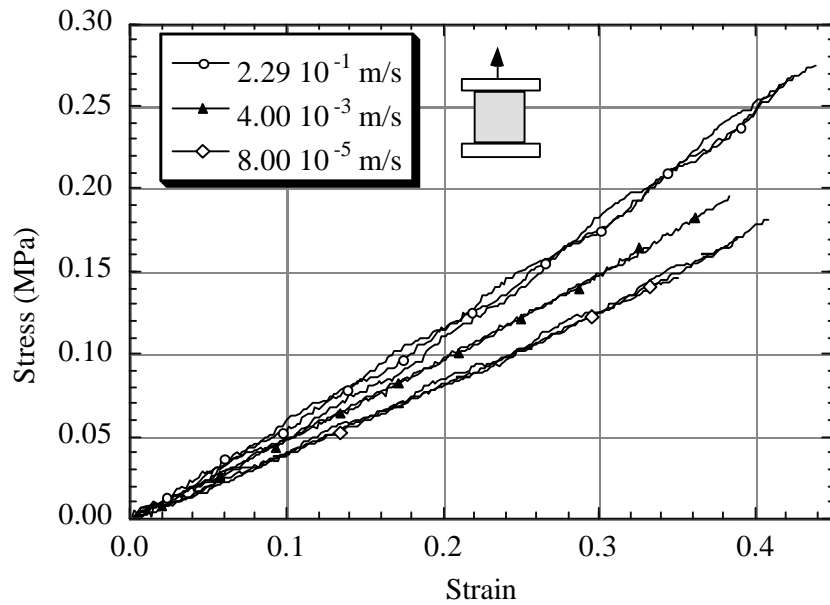
Polypropylene foam (3.06 pcf)

# Shear Response of Rigid Polystyrene Foam

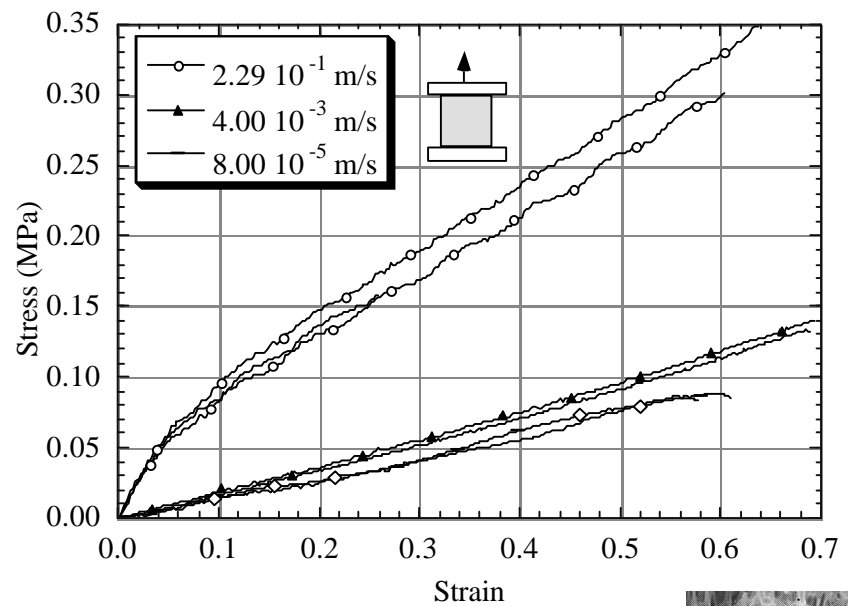
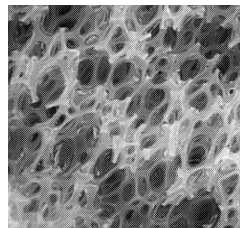


Polystyrene Foam (1.0 pcf) under shear loading

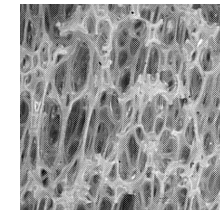
# Tensile Response of Polyurethane Foams



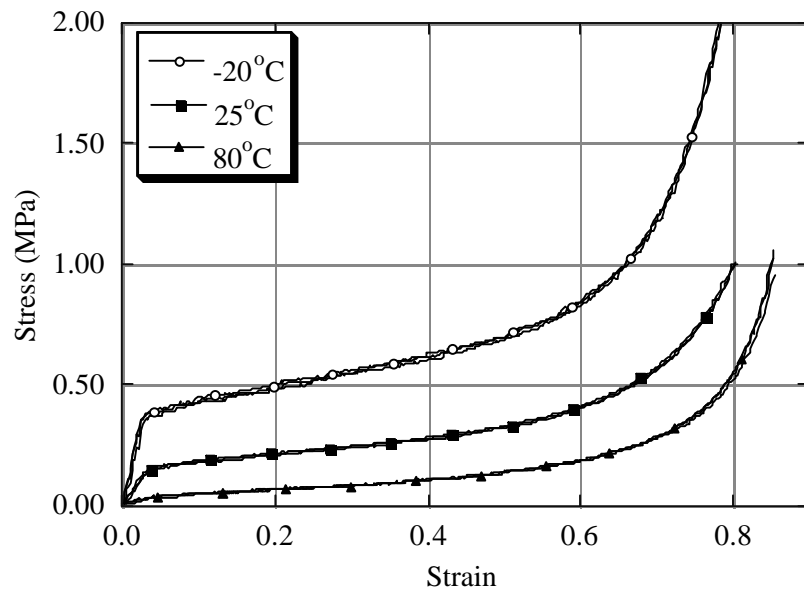
PU Foam (4.3 pcf)



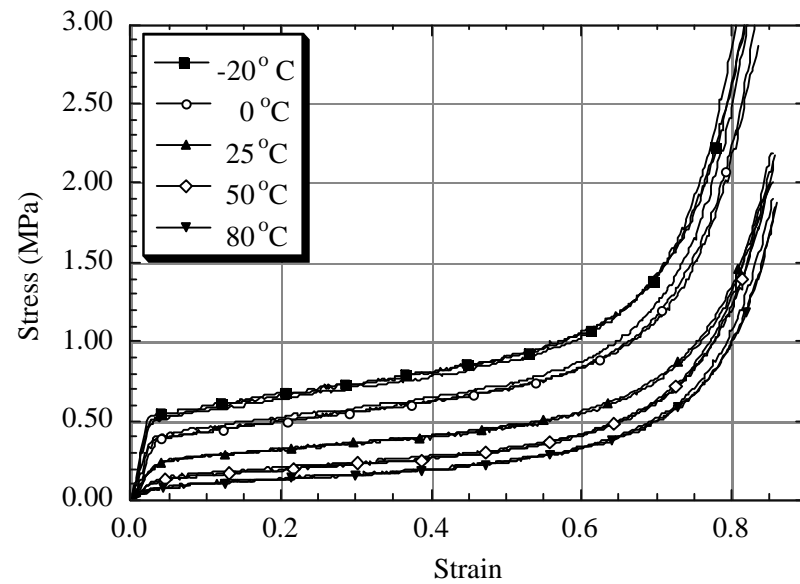
PU Foam (6.0 pcf)



# Temperature Effect on Polypropylene Foam



Strain rate 0.0016 1/sec



Strain rate 4.6 1/sec

Polypropylene Foam (3.06 pcf) under Uniaxial Compression

# Rigid Polymeric Foam Elasticity

## Foam Elasticity

$\dot{\mathcal{S}} = C : \dot{\mathcal{E}}$  where objective stress rate  $\dot{\mathcal{S}}_J^{\mathcal{N}}$  is the Jaumman stress rate in a corotational frame

Isotropic Foam  $\dot{\mathcal{S}} = 2G(\dot{\mathcal{E}}_d - \dot{\mathcal{E}}_{dp}) - K(\dot{\mathcal{E}}_v - \dot{\mathcal{E}}_{vp})$

## Anisotropy Foam

$$\mathcal{E} = \mathcal{S} : \mathcal{S}$$

$$\mathcal{S} = \begin{bmatrix} \frac{1}{E_1} & -\frac{\nu_{21}}{E_2} & -\frac{\nu_{31}}{E_3} & 0 & 0 & 0 \\ -\frac{\nu_{12}}{E_1} & \frac{1}{E_2} & -\frac{\nu_{32}}{E_3} & 0 & 0 & 0 \\ -\frac{\nu_{13}}{E_1} & -\frac{\nu_{23}}{E_2} & \frac{1}{E_3} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{G_{23}} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{G_{13}} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{G_{12}} \end{bmatrix}$$

# Yield Locus for Rigid Polymeric Foam

## Foam Yield Locus

### Isotropic elasto-plastic foam

Dimensional Argument (Gibson and Ashby, 1988)

$$M = \frac{\sigma_{ys} b t^2}{4} \left[ 1 - \left( \frac{\sigma_a}{\sigma_{ys}} \right)^2 \right] \Rightarrow \sigma_{vm} = \kappa \left[ 1 - \left( \frac{p}{\beta} \right)^2 \right]$$

$$P_{crit} = \frac{n^2 p^2 E_s I}{h^2}$$

### Proposed yield locus

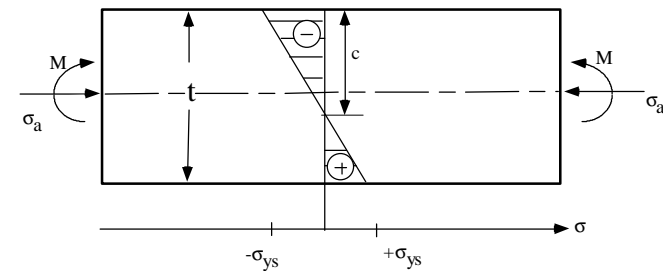
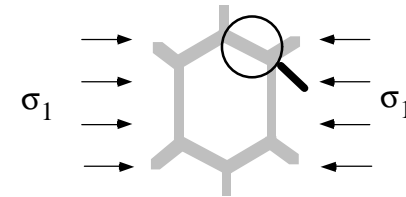
$$\Rightarrow F(S) - F_0 = \frac{[p - x_0(\epsilon_{vp})]^2}{a(\epsilon_{vp})} + \frac{\sigma_{vm}^2}{b(\epsilon_{vp})} - 1 = 0$$

### Anisotropy elasto-plastic foam

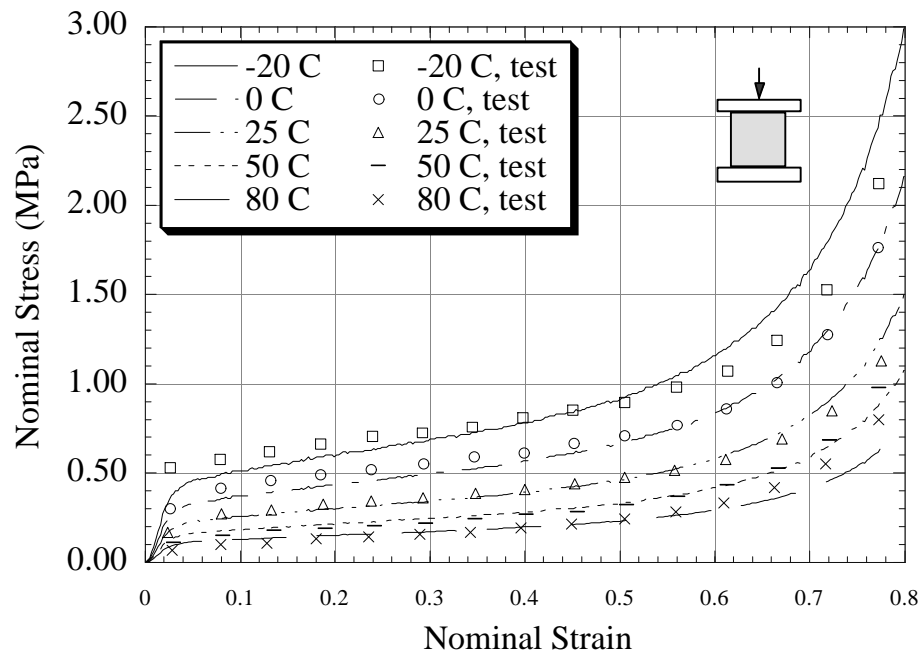
$$F(S) = \sqrt{\tilde{J}} + a\tilde{I} - 1$$

$$\tilde{J}(S) = \frac{1}{2} \left[ \left( \frac{\sigma_{11}}{k_{11}} - \frac{\sigma_{22}}{k_{22}} \right)^2 + \left( \frac{\sigma_{22}}{k_{22}} - \frac{\sigma_{33}}{k_{33}} \right)^2 + \left( \frac{\sigma_{33}}{k_{33}} - \frac{\sigma_{11}}{k_{11}} \right)^2 \right] + 3 \left[ \left( \frac{\sigma_{12}}{k_{12}} \right)^2 + \left( \frac{\sigma_{23}}{k_{23}} \right)^2 + \left( \frac{\sigma_{31}}{k_{31}} \right)^2 \right]$$

$$\tilde{I}(S) = \frac{\sigma_{11}}{k_{11}} + \frac{\sigma_{22}}{k_{22}} + \frac{\sigma_{33}}{k_{33}}$$



# Temperature Sensitivity



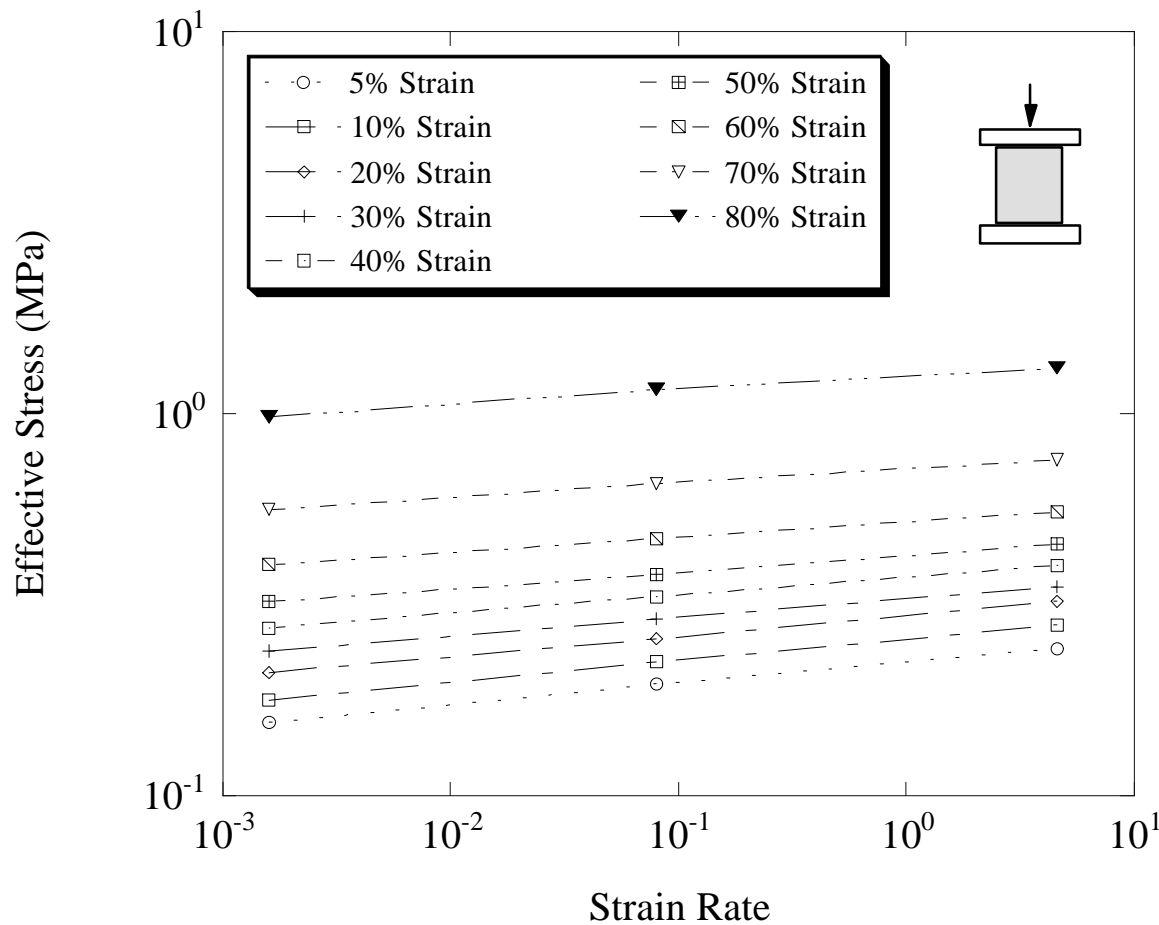
Williams-Landel-Ferry (WLF) Equation  
(Williams et al, 1955)

$$L(T) = \exp \left[ -\frac{C_1(T - T_r)}{C_2 + T - T_r} \right]$$

PP foam (3.06 pcf)

$$C_1 = 6.52 \text{ } ^\circ\text{C}, C_2 = 468.7 \text{ } ^\circ\text{C}$$

# Rate Dependency of PP foam (3.06 pcf)



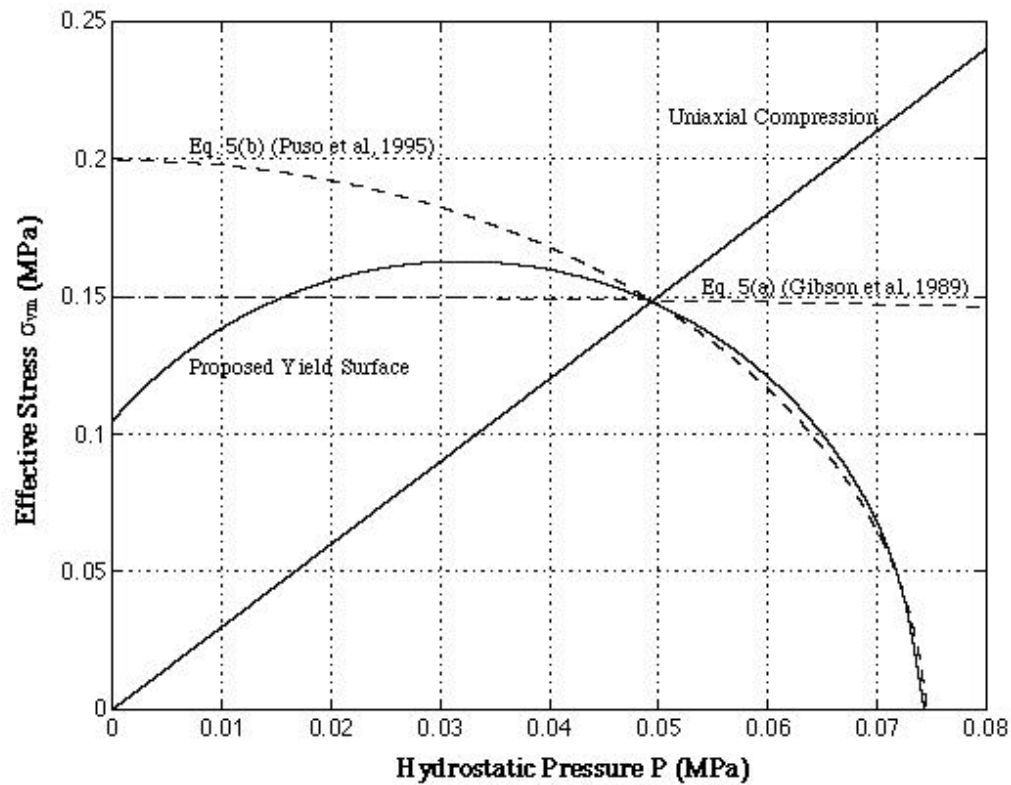
Nagy et al, 1964

$$\dot{\bar{\epsilon}}_p = D \left( \frac{f}{f_0} \right)^{\frac{1}{n}} \quad n = a + b\bar{\epsilon}_p$$

Combined temperature and rate effect

$$\sigma(\epsilon) = \sigma_0(\epsilon)L(T) \left( \frac{\dot{\bar{\epsilon}}}{\dot{\bar{\epsilon}}_0} \right)^{a+b\epsilon}$$

# Comparison of Yield Criterion



Plastic yield envelop (Gibson et al, 1989)

$$\frac{\sigma_{vm}}{\sigma_{ys}} = -\gamma \frac{p}{\rho_{st}} \left( 1 - \frac{3p}{\sigma_{ys} \rho_{st}} \right)$$

Buckling surface (Puso and Govindjee 1995)

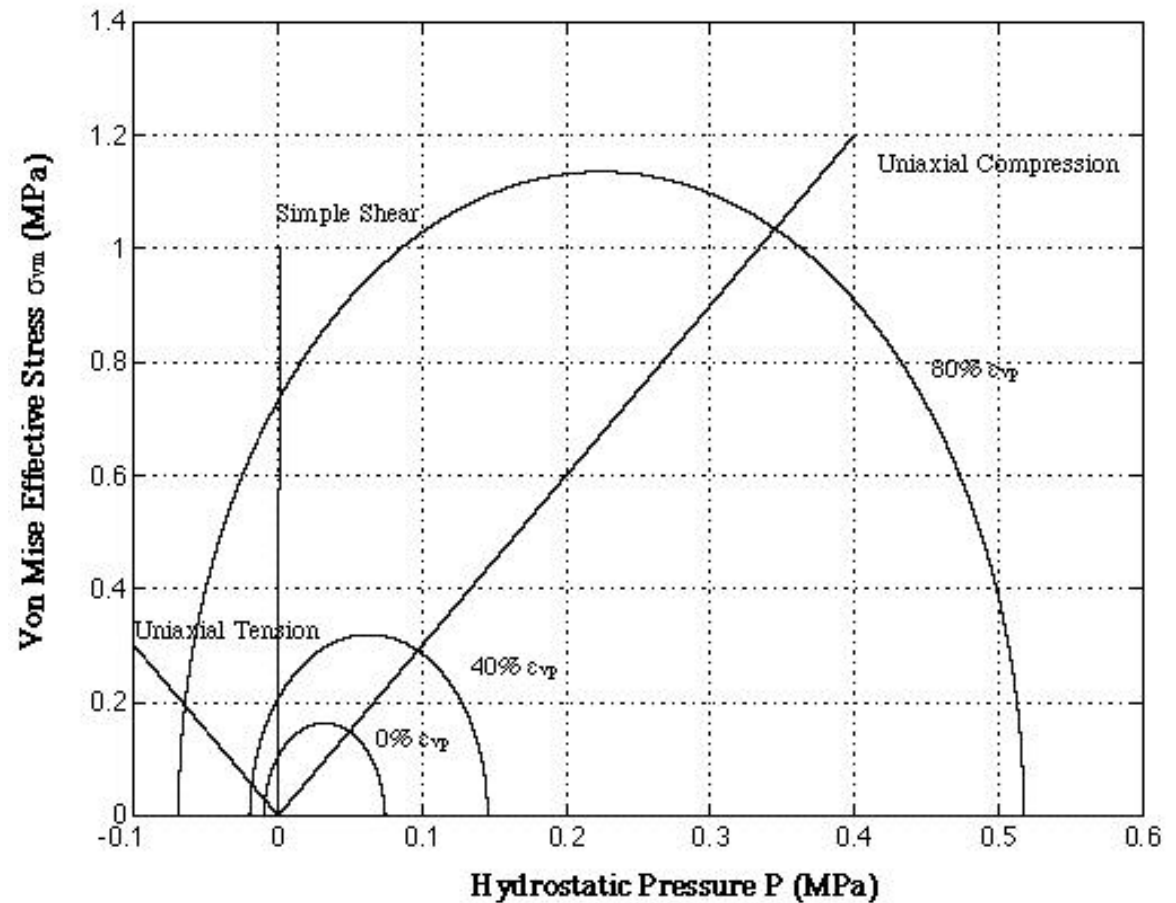
$$\sigma_{vm}^2 + \frac{1}{R^2} (p^2 - h^2) = 0$$

# Kinematic Hardening of Polypropylene Foam (3.06 pcf)

## Kinematic hardening

$$F = F(s, e_{vp}, \mathcal{X})$$

$$g = g(s, e_{vp}, \mathcal{X})$$



**Evolution of Yield Ellipse with Plastic Volumetric Strain**

# Stress Integration Procedure for Elastic-plastic Materials

Deformation decomposition

$$de = de^e + de^p$$

Plasticity consistency condition

$$de = \mathbf{D}^{-1} d\mathbf{s} + \frac{\partial g}{\partial \mathbf{s}} d\lambda$$

$$\left\{ \frac{\partial F}{\partial \mathbf{s}} \right\} d\mathbf{s} - A d\lambda = 0$$



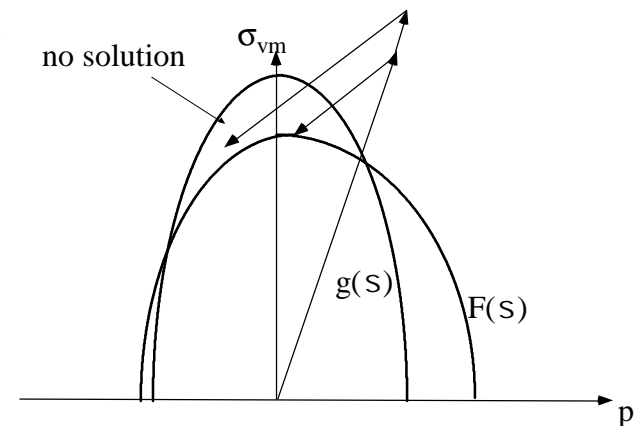
$$\begin{Bmatrix} \{e\} \\ 0 \end{Bmatrix} = \begin{bmatrix} \mathbf{D}^{-1} & \left\{ \frac{\partial g}{\partial \mathbf{s}} \right\} \\ \left\{ \frac{\partial F}{\partial \mathbf{s}} \right\}^T & -A \end{bmatrix} \begin{Bmatrix} \{s\} \\ d\lambda \end{Bmatrix}$$

$$d\mathbf{s} = \mathbf{D}_{ep}^* de$$

$$\mathbf{D}_{ep}^* = \mathbf{D} - \mathbf{D} \begin{Bmatrix} \frac{\partial g}{\partial \mathbf{s}} \\ \frac{\partial F}{\partial \mathbf{s}} \end{Bmatrix} \begin{Bmatrix} \frac{\partial F}{\partial \mathbf{s}} \end{Bmatrix}^T \mathbf{D} \left[ A + \begin{Bmatrix} \frac{\partial F}{\partial \mathbf{s}} \end{Bmatrix}^T \mathbf{D} \begin{Bmatrix} \frac{\partial g}{\partial \mathbf{s}} \end{Bmatrix} \right]^{-1}$$

if  $F \neq g$   $\mathbf{D}_{ep}^*$  is a non-symmetric matrix

Non-unique solution for non-associative plastic flow  
The stress return is not radial



# Non-smooth Multisurface Plasticity

Plastic potential variation (assuming associative plastic potential)

$$\dot{\lambda}_i = \frac{\partial F_i}{\partial s} : C : e - \lambda \frac{\partial F_i}{\partial s} : C : s = 0 \quad (i=1,2,\dots)$$

If plastic yield and loading condition active

$$F_i(s) = 0 \quad \text{and} \quad \frac{\partial F_i}{\partial s} : C : e > 0$$

Plasticity consistency condition

$$\dot{\lambda}_i = \frac{\partial_s F_i : C : e}{\partial_s F_i : C : s}$$

(1)  $\dot{\lambda}_i = 0$  ( $i=1,2,\dots$ ), loading is not active;

(2)  $\dot{\lambda}_i > 0$   $\dot{\lambda}_i = \partial_s F_i : C : e / \partial_s F_i : C : s$

(3)  $\dot{\lambda}_i > 0$  for multiple surfaces  $\lambda = \max(\partial_s F_i : C : e / \partial_s F_i : C : s, i=1,2,\dots)$

Closest-point-projection (Simo et al, 1988)

In summary

$$C^{ep} = \begin{cases} C & \text{if } \dot{\lambda} = 0 \\ C - \frac{[C : s] \otimes [C : \partial_s F_i]}{\partial_s F_i : C : s} & (i=1,2,\dots) \text{ if } \dot{\lambda} > 0 \end{cases}$$

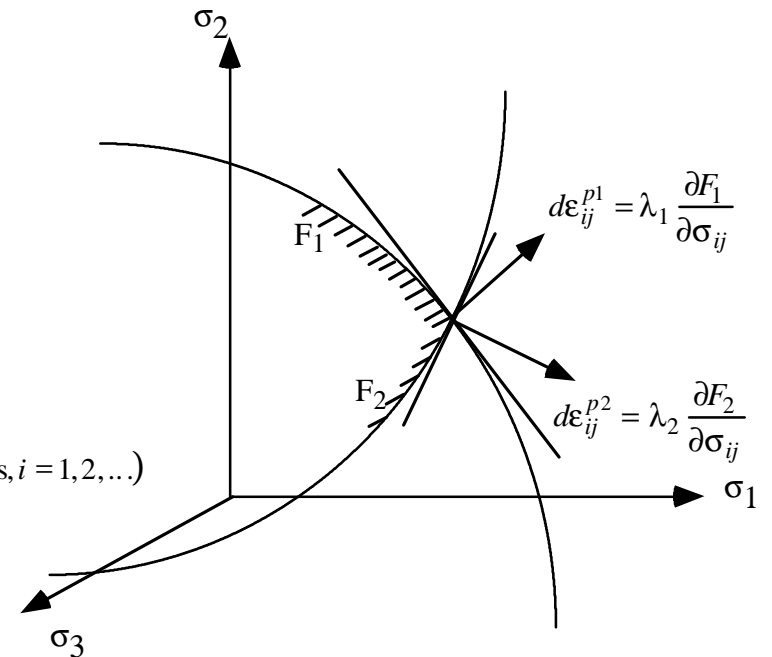
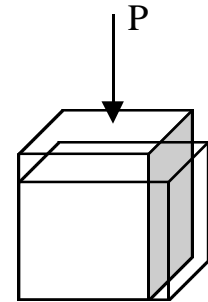


Illustration of a singular point in yield Surface

# Polymeric Rigid Foam Plastic Flow Law

- ➔ Non-associative plastic potential  $g(\hat{s}, \phi) = \sqrt{\alpha p^2 + \sigma_{vm}^2}$
- Plasticity consistency condition  $\dot{\tilde{\epsilon}}_p = \dot{\lambda} \frac{\partial g(\hat{s}, \phi)}{\partial \hat{s}}$
- Plastic Poisson's Ratio  $\dot{\epsilon}_{xyp} = \dot{\epsilon}_{yyp} = -\nu_p \dot{\epsilon}_{zyp}$
- Under uniaxial compression  $\dot{\epsilon}_{vp} = (1 - 2\nu_p) \dot{\epsilon}_{zyp}$



$$\dot{\tilde{\epsilon}}_p = \dot{\lambda} \frac{1}{2g} \left[ 2\sigma_{vm} \frac{\partial \sigma_{vm}}{\partial \hat{s}} + 2\alpha p \frac{\partial p}{\partial \hat{s}} \right] \quad \text{or} \quad \dot{\tilde{\epsilon}}_p = \dot{\lambda} \frac{3}{2g} \left[ \hat{s} - \frac{2\alpha}{9} p \hat{\mathbf{I}} \right]$$

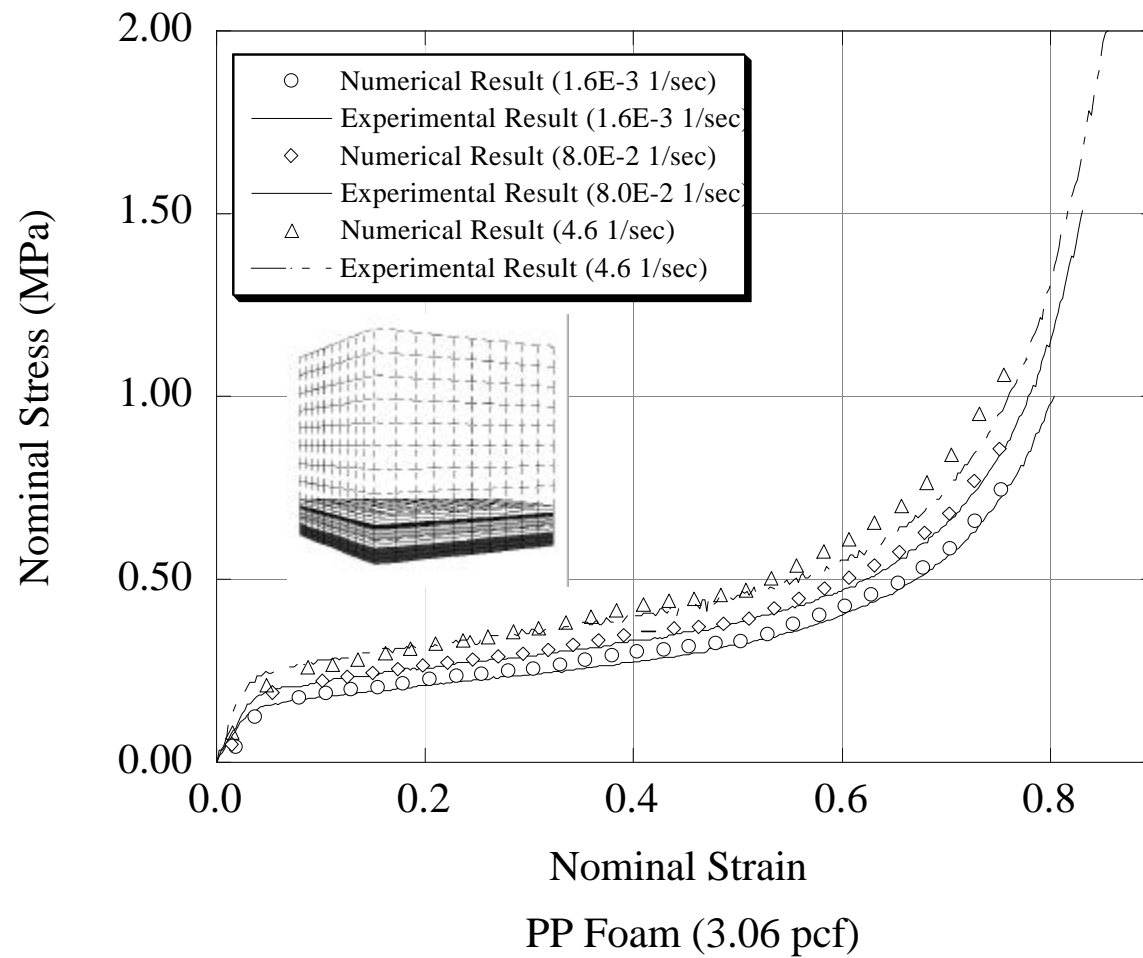
$$\dot{\epsilon}_{zyp} = \dot{\lambda} \frac{3}{2g} \left( s_{zz} - \frac{2\alpha}{9} p \right)$$

$$\dot{\epsilon}_{vp} = -\dot{\lambda} \frac{\alpha p}{g}$$

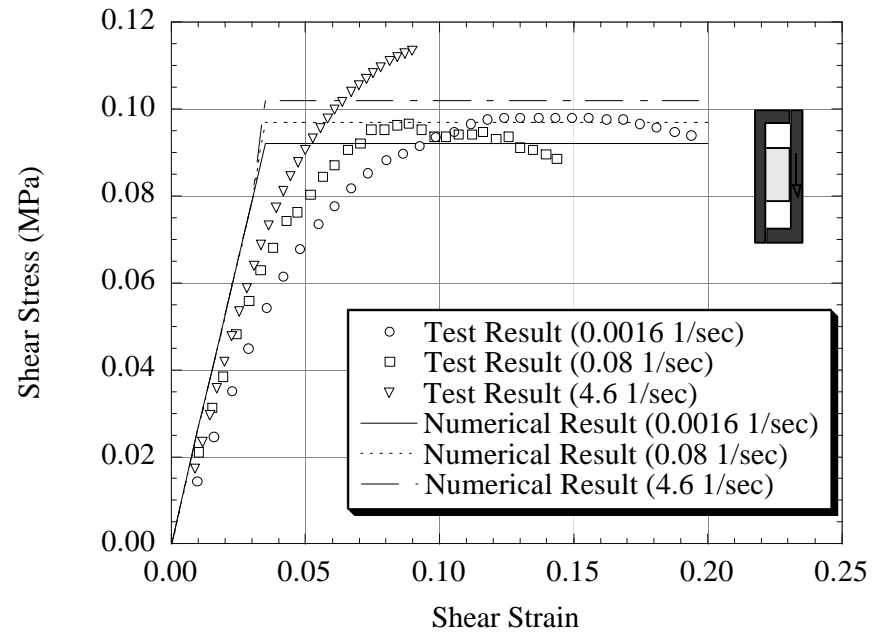
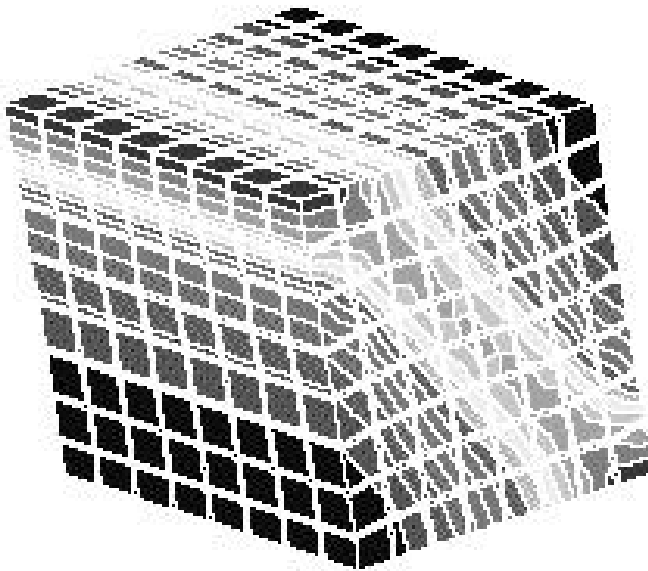
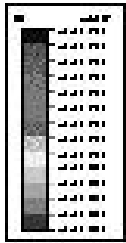
$$\alpha = \frac{9(1 - 2\nu_p)}{2(1 + \nu_p)} \quad \text{zero plastic Poisson's ratio} \quad \rightarrow$$

$$g = \sqrt{\frac{9}{2} p^2 + \sigma_{vm}^2}$$

# Model Validation under Uniaxial Compression

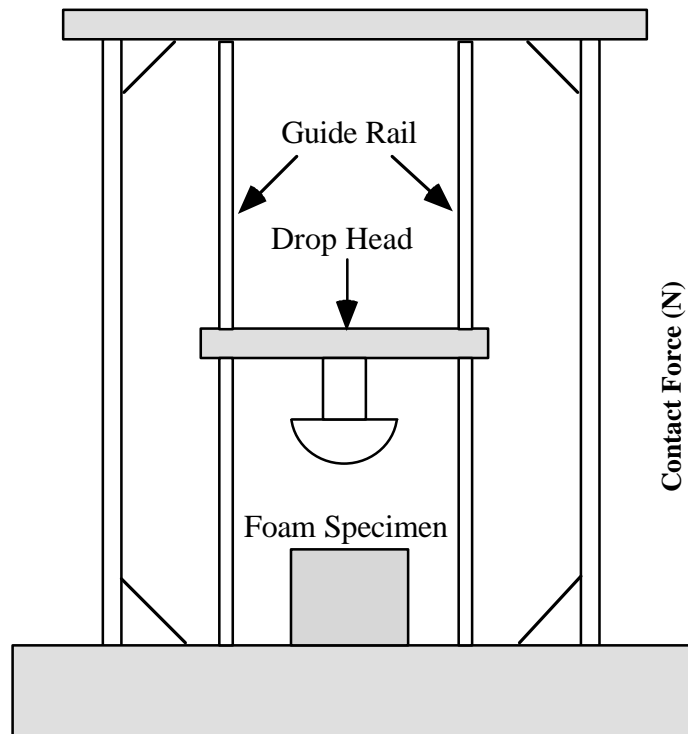


# Model Validation under Simple Shear

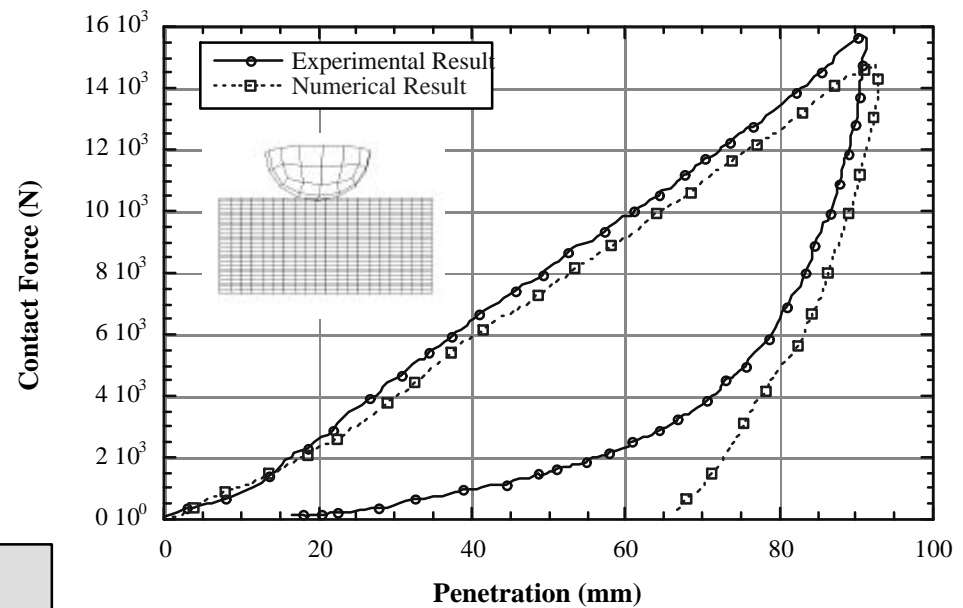


Polystyrene Foam (1.0 pcf)

# Hemispherical Free Drop Test

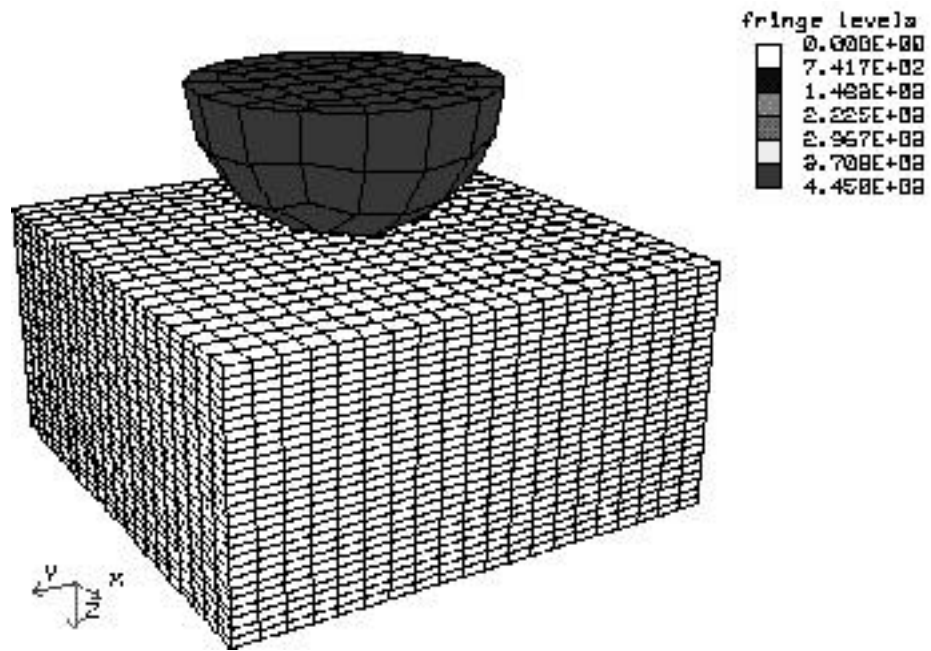


Free Drop Machine

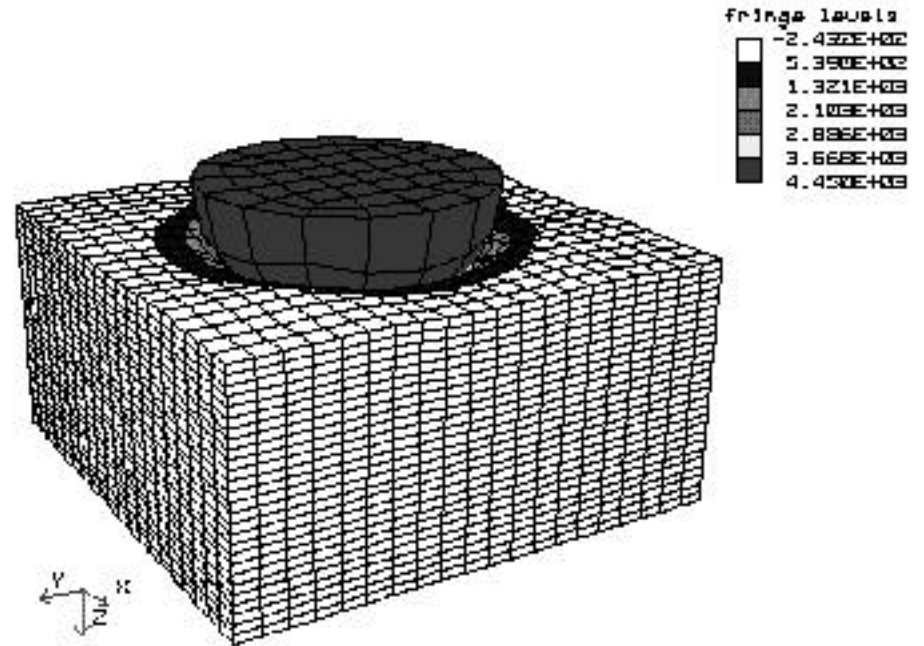


Indenter:  $\phi 127$  mm, 22.2 kgm, 4.5m/sec  
PP foam, 203x 203 x101 mm<sup>3</sup>, 3.06 pcf

# Hemispherical Free Drop Numerical Simulation

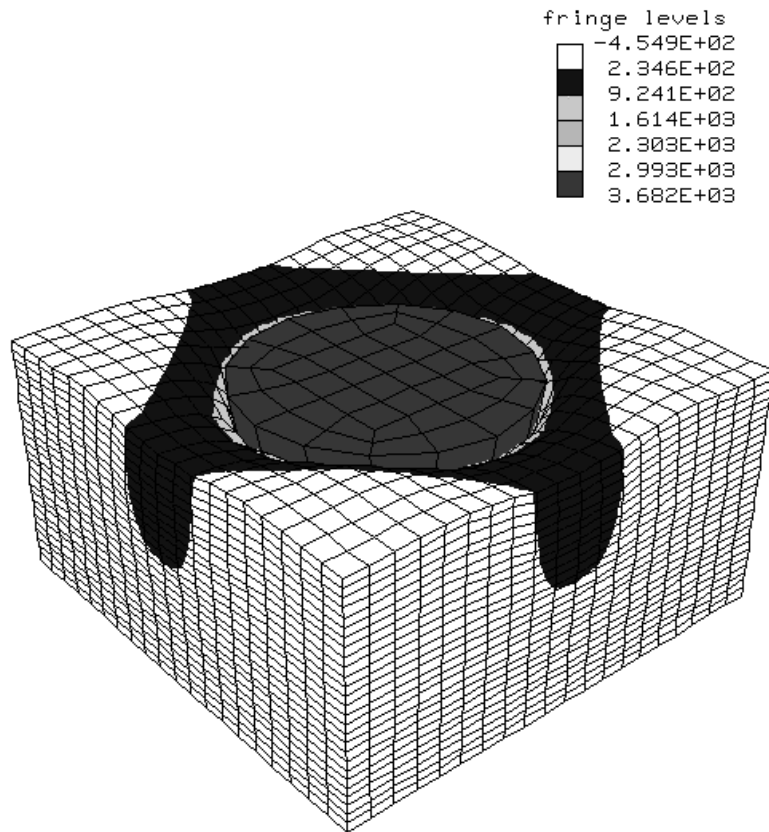


Original Mesh

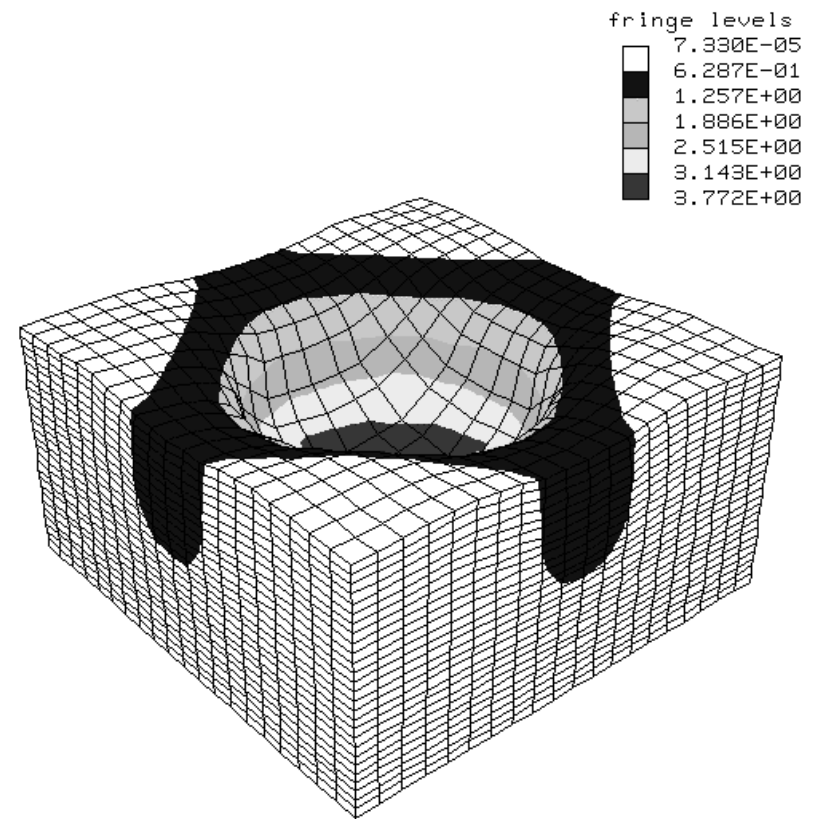


Deformed Mesh at t=9.0 ms

# Hemispherical Free Drop Numerical Simulation



Deformed Mesh at t=16.0 ms



Effective plastic strain t=16.0 ms

## Conclusion

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- PU foams are flexible while PS and PP foams are rigid at 20° C;
- Yield stress of polymeric foams are sensitive to strain rate, temperature and pressure;
- A phenomenological rate dependent single surface elasto-plastic yield criterion is developed and implemented in LS-DYNA3D program;

## Future Work

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- Experiment on polymeric foams under multiaxial loading;
- Constitutive modeling considering different failure mechanism;
- Validate rigid foam model under multiaxial loading;
- Couple homogenization constitutive modeling and LS-DYNA3D;
- Three-dimensional RVE modeling and analysis by using more powerful CT scanner.
- Three-dimensional foam design optimization.